

TITLE OF THE INVENTION
MOVABLE STAGE APPARATUS

FIELD OF THE INVENTION

5 The present invention relates to a lithography technique for manufacturing semiconductor devices or the like and, more particularly, to a technique for projecting a reticle pattern onto a limited band-like rectangular or arcuate region on a wafer and scanning a
10 reticle and the wafer synchronously in one-dimensional direction, so that the reticle pattern is transferred onto the wafer.

BACKGROUND OF THE INVENTION

15 Conventionally, the photolithography process as one of processes for manufacturing semiconductor devices represented by semiconductor integrated circuits uses various types of exposure apparatuses to expose a circuit pattern or the like formed on a mask
20 or reticle (to be referred to as a reticle hereinafter) onto a wafer coated with a photosensitive agent (photoresist).

 Currently, as the semiconductor devices shrink in feature size and increase in integration density, a
25 projection exposure apparatus or so-called stepper, and a so-called scanning type exposure apparatus have been widely used. The projection exposure apparatus moves

the irradiation region of the exposure light to expose the exposure regions of a reticle. The scanning type exposure apparatus projects a reticle pattern onto a limited band-like rectangular or arcuate region on a wafer, and scans a reticle and the wafer synchronously in one-dimensional direction, so that the reticle pattern is transferred onto the wafer. The movable stage apparatus described above generally uses linear motors as its driving source. Fig. 6 shows a conventional movable stage apparatus (Japanese Patent Laid-Open No. 10-12539) mounted on the scanning type projection exposure apparatus described above.

The conventional movable stage apparatus is constituted by a guide 102 fixed on a reticle base 101, a reticle stage 103 reciprocally movable in a Y-axis direction (the scanning direction of the reticle stage will be defined as the Y-axis; to be referred to as the Y-axis direction hereinafter) along the guide 102, and a pair of linear motors 106a and 106b arranged on the two sides of the reticle stage 103 along its traveling path to accelerate and decelerate the reticle stage 103 in the Y-axis direction. The reticle stage 103 is guided by a hydrostatic bearing (air slide) 107 and a preload mechanism in noncontact with a guide surface.

A reticle R is chucked on the reticle stage 103, and a projection optical system (not shown) is provided under the reticle stage 103. A wafer is held by a

wafer stage (not shown). The wafer stage also has a driving mechanism similar to that of the reticle stage 103, and is controlled in the same manner as the reticle stage 103.

5 In general, the resolution of an exposure apparatus is mainly determined by a wavelength λ of an exposing illumination light source and a numerical aperture NA of a projection optical system. More specifically, the shorter the wavelength λ of the
10 exposing illumination light source to be used or the larger the numerical aperture NA of the projection optical system, the higher the resolution. Therefore, the wavelength of the exposing illumination light source used in the exposure apparatus decreases year by
15 year, and the numerical aperture NA of the projection optical system increases.

 In a current mainstream exposure apparatus, a KrF excimer laser ($\lambda = 248$ nm) or ArF excimer laser ($\lambda = 193$ nm) is used as the exposing illumination light
20 source. An exposure apparatus that uses an F_2 laser with a shorter wavelength ($\lambda = 157$ nm) is also being introduced on a practical level.

 To transfer a circuit pattern with a practical minimum line width of 70 nm to 100 nm or less onto a
25 wafer, a light source with a much shorter wavelength is necessary. To meet the above demand, an EUV exposure apparatus that uses EUV (Extreme Ultraviolet) light

with a wavelength λ of about 13 nm, which is shorter than even that of the exposing illumination light source described above by one or more orders of magnitude, has been attracting attention.

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SUMMARY OF THE INVENTION

A reticle stage as a master stage for an exposure apparatus that uses EUV light as the exposure light has the following problems to be solved.

10 EUV light is absorbed by materials greatly. Hence, on a reticle or mask as a transmitting master, the exposure energy reaching a wafer as a photosensitive substrate decreases greatly due to light absorption by the master. For this reason, in the EUV
15 exposure apparatus, use of a reticle or master as a reflecting master has been considered.

When a reflecting master is mounted on a conventional stage as shown in Fig. 6, as the master is irradiated with EUV light from a lower portion in
20 Fig. 6, an opening through which the EUV light passes must be formed in the guide. This makes an EUV optical system complicated, and accordingly a guide or motor should not interfere with the EUV light. This is not preferable as an exposure apparatus.

25 The present invention has been made to solve these problems, and has as its object to provide a preferable stage apparatus on which a reflecting master

is to be mounted. It is another object of the present invention to provide an EUV exposure apparatus which can suitably use a reflecting master.

In order to achieve the above objects, according to the first aspect of the present invention, a movable stage apparatus is comprising a master stage on which a reflecting master is to be mounted, wherein when a space is divided by a plane including a reflection surface of the master, a guide surface to guide movement of the master stage is arranged in a space opposite to a space where an exposure light beam to be reflected by the master passes.

According to the second aspect of the present invention, an exposure apparatus is comprising the movable stage apparatus described above.

According to the third aspect of the present invention, a device manufacturing method is including steps of applying a photosensitive material to a substrate, transferring a pattern by the exposure apparatus described above to the photosensitive material on the substrate coated with the photosensitive material, and developing the substrate on which the pattern has been transferred.

Other features and advantages besides those discussed above shall be apparent to those skilled in the art from the description of a preferred embodiment of the invention which follows. In the description,

reference is made to accompanying drawings, which form a part hereof, and which illustrate an example of the invention. Such example, however, is not exhaustive of the various embodiments of the invention, and therefore
5 reference is made to the claims which follow the description for determining the scope of the invention.

BRIEF DESCRIPTION OF THE DRAWINGS

Fig. 1 is a view showing the schematic
10 arrangement of a movable stage apparatus according to the first embodiment;

Fig. 2 is a schematic view showing the relationship between a coarse movement stage and fine movement stage in the movable stage apparatus according
15 to the first embodiment;

Fig. 3A is a view showing a section taken along the line a - b of Fig. 2;

Fig. 3B is a view showing a section taken along the line b - b of Fig. 2;

20 Fig. 4 is a view showing the schematic arrangement of a scanning type exposure apparatus which uses a movable stage apparatus according to the first or second embodiment;

Fig. 5 is a view showing the schematic
25 arrangement of the movable stage apparatus according to the second embodiment;

Fig. 6 is a view showing the schematic

arrangement of a conventional movable stage apparatus;
and

Fig. 7 is a flow chart showing the flow of an
overall semiconductor device manufacturing process.

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DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

The preferred embodiments of the present
invention will be described.

First, the outline of the preferred embodiments
10 will be described.

A movable stage apparatus according to an
embodiment of the present invention is characterized in
that, when the space is divided by a plane including
the reflection surface of a reticle as a master, a
15 guide surface is formed in a space opposite to a space
where an exposure light beam passes.

The movable stage apparatus desirably has a
coarse/fine movement separate type stage including a
coarse movement stage which reciprocally moves on a
20 guide surface formed on a base in the scanning
direction along the guide in a noncontact manner, and a
fine movement stage having a 6-axis alignment mechanism.

Desirably, the stators of linear motors as thrust
generating mechanisms for the coarse movement stage and
25 fine movement stage are supported in noncontact with
the base and are driven in a noncontact manner. The
linear motor stators desirably have counter mass

mechanisms that move in a direction opposite to the driving direction of the coarse movement stage when the coarse movement stage is driven.

Desirably, the fine movement stage is supported
5 in noncontact with the coarse movement stage, and has an electromagnet which transmits a force, generated upon acceleration or deceleration of the coarse movement stage, to the fine movement stage in noncontact with it.

10 Desirably, the fine movement stage is supported in noncontact with the coarse movement stage by a 6-axis alignment mechanism which can perform 6-axis alignment control of the coarse movement stage by a fine movement linear motor, and by a self weight
15 support mechanism which supports the weight of the fine movement stage, and is driven and aligned in a noncontact manner.

When alignment control of the fine movement stage by the 6-axis alignment mechanism is not performed, the
20 fine movement stage may drop due to its own weight. Hence, a magnetic force generated by the self weight support mechanism desirably, constantly attracts the fine movement stage to the coarse movement stage with a force larger than the self weight of the fine movement
25 stage. When the fine movement stage is attracted to the coarse movement stage by the magnetic force, the position and posture of the fine movement stage are

desirably held by an abutting section by means of kinematic coupling in which three spherical bodies engage with a circular conical groove, V-groove, and flat surface, respectively.

- 5 Alignment by means of kinematic coupling is desirably used as initial alignment of a laser interferometer or the like.

Desirably, a mounted interferometer for measuring a Z position, ω_x position (rolling), and ω_y position
10 (pitching), a mounted interferometer for measuring a Y position and ω_z position (yawing), and a mounted interferometer for measuring an X position are mounted on a fine movement stage top plate, and a long mirror for measuring the Z position, ω_x position (rolling),
15 and ω_y position (pitching), a short mirror for measuring the Y position and ω_z position (yawing), and a long mirror for measuring the X position are mounted on a base vibration-insulated from a base that supports a reticle stage.

- 20 An exposure apparatus that has the movable stage described above, and the manufacture of semiconductor devices and display devices using the exposure apparatus are also within the scope of the present invention.

- 25 In the embodiments constituted as described above, when the space is divided by a plane including the reticle reflection surface, the guide surface of the

reticle stage where a reflecting reticle is to be mounted is formed in a space opposite to a space where the exposure light beam passes. Thus, the optical system can be disposed freely and easily. Since the
5 guide member of the reticle stage need not have an opening where the exposure light beam passes, an increase in rigidity and accuracy of the guide portion can be realized.

The fine movement stage is supported in
10 noncontact with the coarse movement stage by the self weight support mechanism using magnetic elements and the 6-axis alignment mechanism using the fine movement linear motor. When alignment control by the linear motor is not performed, the fine movement stage may
15 drop due to its own weight. Hence, the magnetic force generated by the self weight support mechanism is constantly set larger than the weight of the fine movement stage, so that heat generated by a Z-axis actuator in the direction of self weight can be
20 decreased, and dropping of the fine movement stage can be prevented.

When the fine movement stage is attracted to the coarse movement stage by the magnetic force, it is abutted against and held by an alignment section by
25 means of kinematic coupling. Thus, this alignment serves as aligning the initial position of the laser interferometer or the like.

A position measuring mirror which may decrease the alignment accuracy of the fine movement stage is formed not on the top plate of the fine movement stage, but on the base which is vibration-insulated from the base that supports the reticle stage and which supports the projection optical system. Thus, an increase in the accuracy of the fine movement stage can be realized.

The embodiments of the present invention will be described in detail.

10 (First Embodiment)

Fig. 1 is a view showing the schematic arrangement of a movable stage apparatus according to the first embodiment of the present invention.

The movable stage apparatus according to this embodiment has a reticle base 1, a flat plate-like guide 2 as a guide means fixed on the reticle base 1, a coarse movement stage 3 reciprocally movable along the guide 2 in the Y-axis direction (the scanning direction and non-scanning direction of the reticle stage are defined as Y- and X-axes, respectively, and will be respectively referred to as the Y- and X-axis directions hereinafter), a pair of linear motor stators 4a and 4b arranged on the two sides of the coarse movement stage 3 along its traveling path, and a pair of linear motor movable elements 5a and 5b integrally formed with the two side surfaces of the coarse movement stage 3. The linear motor stators 4a and 4b

and the linear motor movable elements 5a and 5b make up a pair of linear motors 6a and 6b as a means for accelerating and decelerating the coarse movement stage 3 in the scanning direction. The linear motors 6a and 5 6b on the two sides are controlled independently of each other, thus enabling alignment in the rotational direction about the optical axis (the direction of the optical axis is defined as the Z-axis direction, and will be referred to as the Z-axis direction 10 hereinafter). A fine movement stage 7 having alignment mechanisms of 6-axis directions, i.e., alignment in X-, Y-, and Z-axis directions and rotations ω_x , ω_y , and ω_z about the X-, Y-, and Z-axes, respectively, is mounted on the coarse movement stage 3 in noncontact 15 with it. In the following description, the integrated body of the coarse movement stage 3 and fine movement stage 7 put together will be called a reticle stage.

A reticle R is held by a reticle chuck 8 on the top plate of the fine movement stage 7. When the space 20 is divided by a plane including the reticle reflection surface (pattern surface), the driving point of the guide surface 2 of the reticle stage and that of the coarse movement stage 3 are formed in a space opposite to a space where the exposure light beam passes.

25 Below the reticle stage, a wafer W as an exposure target body is held by a wafer stage, as shown in Fig. 4. The wafer stage also has a driving section

similar to that of the reticle stage, and is controlled in the same manner as the reticle stage. Band-like exposure light which irradiates part of the reticle forms an image on the wafer W through a projection optical system constituted by reflection mirrors, and exposes the band-like region of the wafer W, thus transferring a reticle pattern. A mechanism that transfers the entire reticle pattern to the wafer by causing the reticle stage and wafer stage to travel synchronously is identical to that in the conventional projection exposure apparatus. At this time, the traveling positions of the reticle stage and wafer stage are detected by the interferometers, and are fed back to the respective driving sections. Acceleration and deceleration of the reticle stage, and acceleration control of the reticle stage during exposure are the same as those in the conventional case.

The coarse movement stage 3 has, on the guide 2, flat surfaces 2h and 2v serving as a reference when it reciprocally moves. The flat surface 2h is a surface parallel to the X-Y plane. The flat surface 2v is a surface parallel to the Y-Z plane.

The coarse movement stage 3 is supported and guided by a hydrostatic bearing 10a in noncontact with the reference surface 2 formed on the guide side surface. The coarse movement stage 3 also has a preload mechanism for generating an attracting force

between the guide upper surface 2h and side surface 2v.

When a permanent magnet is used as the preload mechanism, the base is limited to a magnetic body.

When a vacuum suction mechanism is used as the preload
5 mechanism, the material of the base is not particularly limited. The noncontact support mechanism is not limited to one using a hydrostatic bearing, but can be a magnetic levitated type support mechanism using the Lorentz force or reactance force.

10 The linear motor stators 4a and 4b arranged on the two sides of the coarse movement stage 3 along its traveling path are supported and guided by a hydrostatic bearing 10 in noncontact with the guide surface formed on the base 1, in the same manner as the
15 coarse movement stage 3 described above, and have counter masses 9a and 9b, respectively.

 With the above arrangement, when the coarse movement stage 3 is driven in the Y-axis (scanning) direction, the linear motor stators 4a and 4b move in a
20 direction opposite to the moving direction of the coarse movement stage 3 due to the reaction force generated upon driving the coarse movement stage 3. At this time, the moving amounts of the linear motor stators 4a and 4b are determined by the ratio in mass
25 of the coarse movement stage 3 to the linear motor stators 4a and 4b. The driving reaction force generated upon driving the coarse movement stage 3 is

thus canceled, so that the main body structure that supports the base 1 does not vibrate largely. As the position of the total barycenter of the coarse movement stage 3 and linear motor stators 4a and 4b does not
5 move in the scanning direction, the main body structure does not deform. The pair of left and right linear motors 6a and 6b are independent of each other. Even when control in the rotational direction about the Z-axis is performed, as the barycenters and the driving
10 points of the counter masses 9a and 9b coincide, no rotational force is generated in the counter masses 9a and 9b, and no reaction force in the rotational direction is transmitted to the structure that the counter masses 9a and 9b support. The linear motors 6a
15 and 6b, i.e., the guide mechanisms of the counter masses 9a and 9b need only transmit a reaction force in the scanning-axis (Y-axis) direction.

The position measurement of the fine movement stage will be described.

20 A mounted interferometer 11z for measuring a Z position, ω_x position (rolling), and ω_y position (pitching), a mounted interferometer (not shown) for measuring a Y position and ω_z position (yawing), and a mounted interferometer 11x for measuring an X position
25 are mounted on the fine movement stage 7, and a long mirror 12z for measuring the Z position, ω_x position, and ω_y position, a short mirror (not shown) for

measuring the Y position and ω_z position (yawing), and a long mirror 12x for measuring the X position are mounted on another base 13 vibration-insulated from the base 1 that supports the reticle stage. Hence, the
5 positions in 6 degrees of freedom of the fine movement stage 7 are measured.

According to the above arrangement, the long mirrors which decrease the positioning accuracy of the fine movement stage 7 by vibration are arranged not on
10 the fine movement stage 7 but on the base 13 that supports the projection optical system vibration-insulated from the base 1 that supports the reticle stage. Thus, an increase in accuracy of the fine movement stage 7 can be realized.

15 The relationship between the coarse movement stage 3 and fine movement stage 7 will be described with reference to Figs. 2 and 3.

Fig. 2 is a plan view of the movable stage apparatus of the embodiment shown in Fig. 1 in a
20 direction perpendicular to the reticle R. Fig. 3A is a sectional view taken along the line a - b of Fig. 2, and Fig. 3B is a sectional view taken along the line b - b of Fig. 2.

In the movable stage apparatus according to this
25 embodiment, self weight support mechanisms 21 which support a self weight 27 of the fine movement stage 7, 6-axis alignment mechanisms 22, 23, and 24,

electromagnets 25 which transmit the forces generated in the linear motors 6a and 6b upon accelerating or decelerating the coarse movement stage 3 in the Y-axis direction, and alignment sections 26 using kinematic coupling are arranged between the fine movement stage 7 and coarse movement stage 3.

The fine movement stage 7 which can be aligned in the 6-axis directions is mounted in noncontact with the coarse movement stage 3 in the direction of gravity (lower side). Since the fine movement stage 7 is supported in noncontact with the coarse movement stage 3, the behavior (vibration) of the coarse movement stage 3 is not transmitted to the fine movement stage 7.

An actuator that performs alignment in the 6-axis directions requires noncontact driving and high-accuracy alignment, and accordingly desirably uses the linear motors 22, 23, and 24. In this case, for the convenience of wiring or the like, coils are desirably provided to stationary sides (coarse movement stage sides) 22a, 23a, and 24a, and permanent magnets are desirably provided to movable sides (fine movement stage sides) 22b, 23b, and 24b.

As the linear motors, the X-direction fine movement linear motors 22 to drive the fine movement stage 7 in the X-axis direction, the Y-direction fine movement linear motors 23 to drive the fine movement stage 7 in the Y-axis direction, and the Z-direction

fine movement linear motors 24 to drive the fine movement stage 7 in the Z-axis direction are provided. According to this embodiment, the X-direction fine movement linear motors 22 include two motors 22, and the Y-direction fine movement linear motors 23 include two motors 23. With this arrangement, the fine movement stage 7 can be moved not only in the directions of the respective axes, but can also be driven in the rotational direction ω_z about the Z-axis. The Z-direction fine movement linear motors 24 include three motors 24. Thus, the fine movement stage 7 can be moved not only in the Z direction, but can also be driven in the tilt directions (ω_x and ω_y).

In this embodiment, 7 fine movement linear motors are used as the 6-axis alignment mechanisms for the fine movement stage 7. However, the present invention is not limited to this arrangement, but can employ any other arrangement as far as it can generate thrusts in 6-axis directions.

The initial position of the fine movement stage 7 with respect to the coarse movement stage 3 is determined by the alignment sections 26 by means of kinematic coupling. The alignment sections 26 include three spherical bodies equidistant in the circumferential direction. The spherical bodies engage with a circular conical groove, V-groove, and flat surface, respectively. Thus, the positions of the fine

movement stage 7 in the 6-axis directions are constrained, and the posture of the fine movement stage 7 with respect to the coarse movement stage 3 is determined.

5 An electromagnetic coupling will be described with reference to Fig. 3A.

 In this embodiment, the fine movement linear motors 22 and 23 in the X- and Y-axis directions for the coarse movement stage 3 serve to align slightly,
10 and generate small thrusts, so that they cannot apply thrusts as they are over a long distance.

 In view of this, E-shaped electromagnets 25a, which transmit forces generated by the linear motors 6a and 6b upon accelerating or decelerating the coarse
15 movement stage 3 in the Y-axis direction to the fine movement stage 7 in a noncontact manner, are arranged at the center of the coarse movement stage 3 to be perpendicular to the Y-axis direction. Magnetic blocks 25b corresponding to the E-shaped electromagnets 25a
20 are provided to the center of the fine movement stage 7. The electromagnets can be of any type as far as they can generate an attracting force for a magnetic body when currents are supplied to the coils of the electromagnets, and are not limited to the E-shaped
25 ones as in this embodiment.

 Only attracting forces are generated between the electromagnets 25a and magnetic blocks 25b regardless

of the directions of the currents supplied to the coils of the electromagnets. When an electromagnet that generates an attracting force in a positive driving direction and an electromagnet that generates an attracting force in a negative driving direction are provided to form a pair, a large attracting force is applied in the positive and negative Y-axis directions, so that the force generated upon acceleration or deceleration may be transmitted to the fine movement stage 7 in noncontact with it.

With the above arrangement, the X- and Y-axis fine movement linear motors 22 and 23 need only perform slight position control of the fine movement stage 7, and any current that may substantially generate heat to pose an issue does not flow through the X- and Y-axis fine movement linear motors 22 and 23.

The self weight support mechanisms 21 will be described with reference to Fig. 3B.

Each self weight support mechanism 21 is constituted by a magnet 21a and magnets 21b the magnetic forces of which act on each other, and yokes 21c. The magnets of the same pole face each other, and thrust forces in the Z-axis direction (forces that lift the fine movement stage 7 toward the coarse movement stage 3) generated between the magnets of the same pole are utilized, so that the weight of the fine movement stage 7 is supported. In order to cancel forces other

than those in the Z-axis direction, the facing magnets are arranged to be symmetrical with respect to the Z-axis.

With the above arrangement, the Z-direction fine movement linear motors 24 need not have a function of supporting the weight of the fine movement stage 7, but only need to perform position control exclusively. Therefore, any current that may substantially generate heat to pose an issue does not flow through the Z-direction fine movement linear motors 24.

The magnetic attracting force in the Z-axis direction generated by the self weight support mechanisms 21 is always larger than the weight 27 of the fine movement stage 7.

The fine movement stage 7 supports the coarse movement stage 3 in noncontact with it by the self weight support mechanisms 21 and the 6-axis alignment mechanisms 22, 23, and 24 by means of linear motors. Hence, when position control of fine movement stage 7 by means of the Z-direction fine movement linear motors 24 is not performed, the fine movement stage 7 may drop due to its own weight.

In view of this, the magnetic attracting force generated by the self weight support mechanisms 21 is set to be always larger than the self weight 27 of the fine movement stage 7. When alignment control of the fine movement stage 7 is not performed, the fine

movement stage 7 is attracted to the coarse movement stage 3 by the magnetic attracting force of the self weight support mechanisms 21.

With the above arrangement, the fine movement stage 7 can be prevented from dropping. Also, when the fine movement stage 7 is to be attracted to the coarse movement stage 3 by the magnetic force, it is abutted against and held by the alignment sections 26 by means of kinematic coupling, so that the alignment sections 26 can perform alignment for the initial positions of the laser interferometers or the like.

(Second Embodiment)

Fig. 5 is a view showing the schematic arrangement of a movable stage apparatus according to the second embodiment of the present invention.

According to this embodiment, a guide surface 2a which supports a coarse movement linear motor stator 5 and counter mass 9, identical to those of the first embodiment, in noncontact with them is arranged below them.

With this arrangement, the weight of the coarse movement linear motor stator 5 of the coarse movement stage and that of the counter mass 9 act on the guide surface 2a as a preload, so that the preload force generated by a permanent magnet or the like can be decreased as compared to the case of the first embodiment.

According to this embodiment, the guide surface 2a which supports the counter mass 9 of the coarse movement stage in noncontact with it is integral with a guide surface 2 that supports a coarse movement stage 3 in noncontact with it. The arrangement of the guide surface 2a which supports the counter mass 9 in noncontact with it is not particularly limited to this. Alternatively, the counter mass 9 may be supported by another structure vibration-insulated from a structure where the guide surface 2, that supports the coarse movement stage 3 in noncontact with it, is supported.

Except for this, the second embodiment is the same as the first embodiment, and a description thereof will accordingly be omitted.

As described above, according to the embodiments described above, the optical system can be disposed freely and easily without interfering with the optical path where the exposure light beam passes. The guide member of the reticle stage need not have an opening where the exposure light beam passes. Therefore, an increase in rigidity and accuracy of the guide portion can be realized.

A semiconductor device manufacturing process which utilizes this exposure apparatus will be described. Fig. 7 is a flow chart showing the flow of an overall semiconductor device manufacturing process.

In step 1 (circuit design), a semiconductor

device circuit is designed. In step 2 (mask fabrication), a mask is fabricated based on the designed circuit pattern.

In step 3 (wafer manufacture), a wafer is
5 manufactured using a material such as silicon. In step
4 (wafer process) called a preprocess, an actual
circuit is formed on the wafer by lithography using the
mask and wafer with the exposure apparatus described
above. In step 5 (assembly) called a post-process, a
10 semiconductor chip is formed from the wafer fabricated
in step 4. This step includes processes such as
assembly (dicing and bonding) and packaging (chip
encapsulation). In step 6 (inspection), inspections
including operation check test and durability test of
15 the semiconductor device fabricated in step 5 are
performed. A semiconductor device is completed with
these processes, and shipped in step 7.

The wafer process of step 4 has the following
steps, i.e., an oxidation step of oxidizing the surface
20 of the wafer, a CVD step of forming an insulating film
on the wafer surface, an electrode formation step of
forming an electrode on the wafer by deposition, an ion
implantation step of implanting ions into the wafer, a
resist processing step of applying a photosensitive
25 agent to the wafer, an exposure step of transferring
the circuit pattern to the wafer after the resist
processing step with the exposure apparatus described

above, a developing step of developing the wafer
exposed in the exposure step, an etching step of
etching portions other than the resist image developed
in the developing step, and a resist stripping step of
5 stripping any unnecessary resist after etching. By
repeating these steps, a multilayered structure of
circuit patterns is formed on the wafer.

As has been described above, according to the
above embodiments, a space including the guide surface
10 of the reticle stage may not interfere with the
exposure light beam.

The present invention is not limited to the above
embodiments and various changes and modifications can
be made within the spirit and scope of the present
15 invention. Therefore, to apprise the public of the
scope of the present invention the following claims are
made.